

# ASME PTC-13 in Action: Practical Approach to Blower System Performance Testing

---

Hiran de Mel, P.E. and Julie Gass, P.E.  
*Keynote Speakers*

Sponsored by

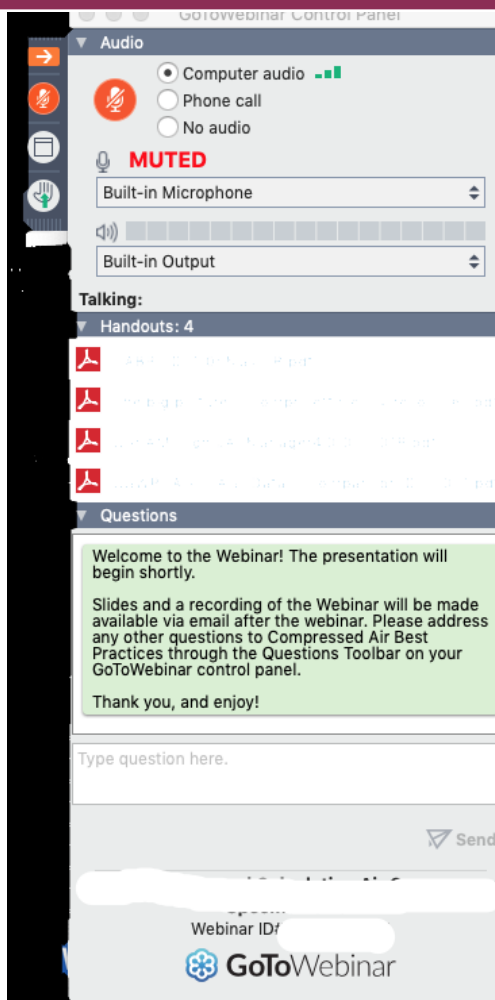


The recording and slides of this webinar will be made available to attendees via email later today.

PDH Certificates will be e-mailed to attendees by within 2 days.



# Q&A Format



- Panelists will answer your questions during the Q&A session at the end of the Webinar.

- Please post your questions in the Questions Window in your GoToWebinar interface.

- Direct all questions to Blower & Vacuum Best Practices® Magazine

Sponsored by



# Handouts

**WASTEWATER** *inovair*  
*Innovative Centrifugal Blowers*

**THE MOST RELIABLE, COST EFFECTIVE AND EFFICIENT BLOWERS**

**LOWEST TOTAL OWNERSHIP COST**  
 inovair's highly efficient and durable geared centrifugal blowers deliver the industry's lowest total ownership cost. The key is a high efficiency design that is also affordable, thanks to the use of industry standard components to avoid unnecessary complexity and cost, as well as the company's nearly 30 years of experience designing and manufacturing compact blowers. inovair's proven design approach also simplifies maintenance and produces extreme durability.

**VERTICALLY INTEGRATED, MADE IN THE USA**  
 With documented energy savings as high as 45% relative to PD and multistage blowers, and without the electrical complexity and durability issues seen in high speed turbo blowers, inovair offers a unique combination of high efficiency, reliability, and simplicity. Unlike some others, inovair blowers and customers are supported by the advantages of a vertically integrated manufacturer, with inovair controlling all elements of design, production, and service. All of these activities are performed 100% in the USA, at the company's headquarters in Kansas City.

**BLOWER OPTIONS FROM 15-600 HP**

**IM-20**  
 25-150 HP

**IM-30**  
 Single or Stacked  
 40-400 HP

**IO SERIES**  
 30-100 HP

**PROUDLY MADE IN USA**

14801 W 114th Terrace, Lenexa, KS 66215 913-469-7244 1-855-INOVAIR inovair.com

The Magazine for Quality & Reliability in Energy-Efficient Blower & Vacuum Systems

**BLOWER & VACUUM BEST PRACTICES**  
 blowervacuumbestpractices.com

**Vacuum Automation**

**AERATION BLOWER SYSTEMS**

14 U.K. Wastewater Treatment Plant Blows Away Inefficiency  
 20 PD Blower Package Reliability

**INDUSTRIAL VACUUM & BLOWER SYSTEMS**

18 The Future of Vacuum Automation for AMRs and AGVs  
 26 Technology News

October 2023

Sustainable, Safe & Reliable On-Site Utilities Powering Automation

**COMPRESSED AIR BEST PRACTICES**  
 airbestpractices.com

**Measurement**

30 Best Practices 2023 EXPO Show Report

12 When to Repair vs. Replace Your Vacuum Pump  
 18 Evapco Celebrates at Global Sales Conference  
 24 Crimping Pipe Nozzles for Improved Efficiency  
 40 Aeration Blowers at Weftec 2023

December 2023

# Disclaimer

---

**All rights are reserved. The contents of this publication may not be reproduced in whole or in part without consent of Smith Onandia Communications LLC. Smith Onandia Communications LLC does not assume and hereby disclaims any liability to any person for any loss or damage caused by errors or omissions in the material contained herein, regardless of whether such errors result from negligence, accident, or any other cause whatsoever.**

**All materials presented are educational. Each system is unique and must be evaluated on its own merits.**

Mark Your Calendars for the Best Practices 2024 EXPO & Conference!

---

# JOIN US IN ATLANTA

October 29-31, 2024 at the Cobb Galleria Centre

## SPEAK AT THE CONFERENCE

**Track 1 – Sustainability Through  
Energy/Water Conservation Measures**

**Track 2 – Quality, Safety and Reliability**

As a Speaker, you will receive complimentary access to the Conference, a \$675 value!

<https://cabpexpo.com/conference/speaker-submission/>

# ASME PTC-13 in Action: Practical Approach to Blower System Performance Testing

Introduction

Blower & Vacuum Best Practices Magazine



Sponsored by

**inovair**<sup>®</sup>  
Geared Centrifugal Blowers

**BLOWER & VACUUM**  
**BEST PRACTICES**  
blowervacuumbestpractices.com

**BEST PRACTICES**  
EXPO & CONFERENCE CABPEXPO.COM  
COMPRESSED AIR / VACUUM / COOLING

# The Speakers



**Hiran de Mel, P.E.**  
Jacobs  
*Keynote Speaker*



**Julie Gass, P.E.**  
Black & Veatch  
*Keynote Speaker*

Sponsored by



# Today's Webinar

---

## Hiran de Mel

- Brief Introduction to ASME PTC 13
- Specifying PTC 13
- Establishing Performance Points
- Developing a Test Plan
- Witness Test Roles and Responsibilities
- Inovair Blower Test 1
- Things Inovair did well and what can be improved

## Julie Gass

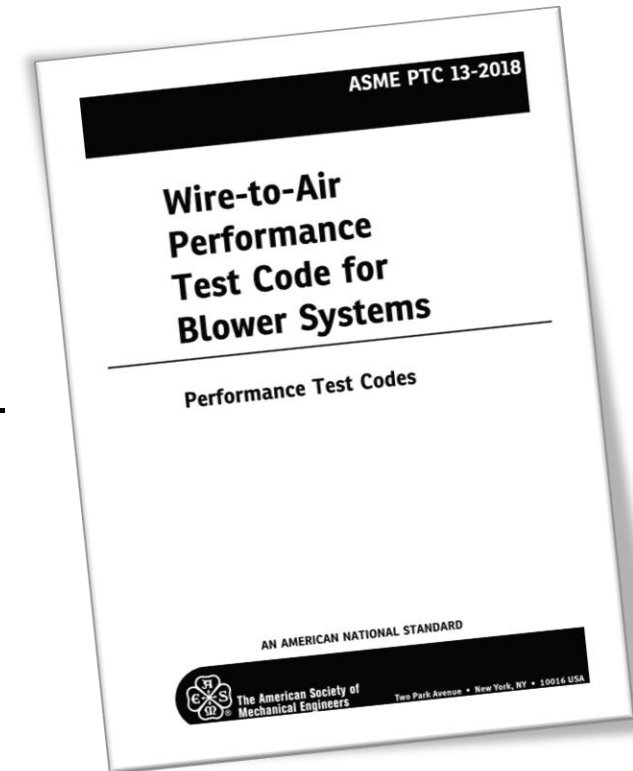
- Inovair Blower Test 2
- Test Protocol submittal
- Witness Testing –How to Conduct & What to Look for
- Test Report and Presentation of Results
- Evaluating and Acceptance of Results
- Things Inovair did Well
- Areas for Improvement
- Summary



# Overview of ASME PTC 13

## "Wire-to-Air Performance Test Code for Blower Systems"

- Developed by: The American Society of Mechanical Engineers (ASME)
- Initiated by: Consortium for Energy Efficiency (CEE) in 2010
- Historical Context:
  - ASME PTC 9 (Displacement Compressors, Vacuum Pumps, Blowers) - First issued in 1915
  - ASME PTC 10 (Compressors and Exhausters) - First issued in 1934
- Purpose: Addresses advancements in blower and compressor technology



# Technological and Industry Relevance

---

- Technological Developments:
  - New mechanisms, control techniques, and integrated systems
- Industry Impact:
  - Focused on wastewater treatment industry
  - Electrical energy usage
- Key Features:
  - Technology-neutral evaluation
  - Wire-to-air power consumption
  - Rigorous, industry-accepted test procedures

# Limitations of ASME PTC 13

---

- Blower Packages with Intercoolers:
  - Not covered; refer to ASME PTC 10
- Blowers with Side Streams:
  - Excluded due to measurement deviations
- Blowers, Fans, and Compressors Applicable when:
  - Gas is air
  - Inlet pressure near atmospheric
  - Pressure ratio  $\leq 3.0$
  - Covers a range of blower types and applications
- Intent:
  - Standardize performance testing for centrifugal and PD blowers
- Testing without adjustment for uncertainty or including a tolerance, yield the best available indication of the actual performance of the tested equipment

# ASME PTC 13 Testing Philosophy - Overview

---

- Objective:
  - Measure "wire-to-air" performance, focusing on electric power consumption under specified conditions.
- Measurement Procedures:
  - Air Capacity: Inlet volumetric flow rate measured at the blower discharge.
  - Inlet Flow Calculation: Based on delivered mass flow and inlet density.
- Performance Verification:
  - Confirm compliance with specified conditions including flow rate, pressure, and environmental factors.

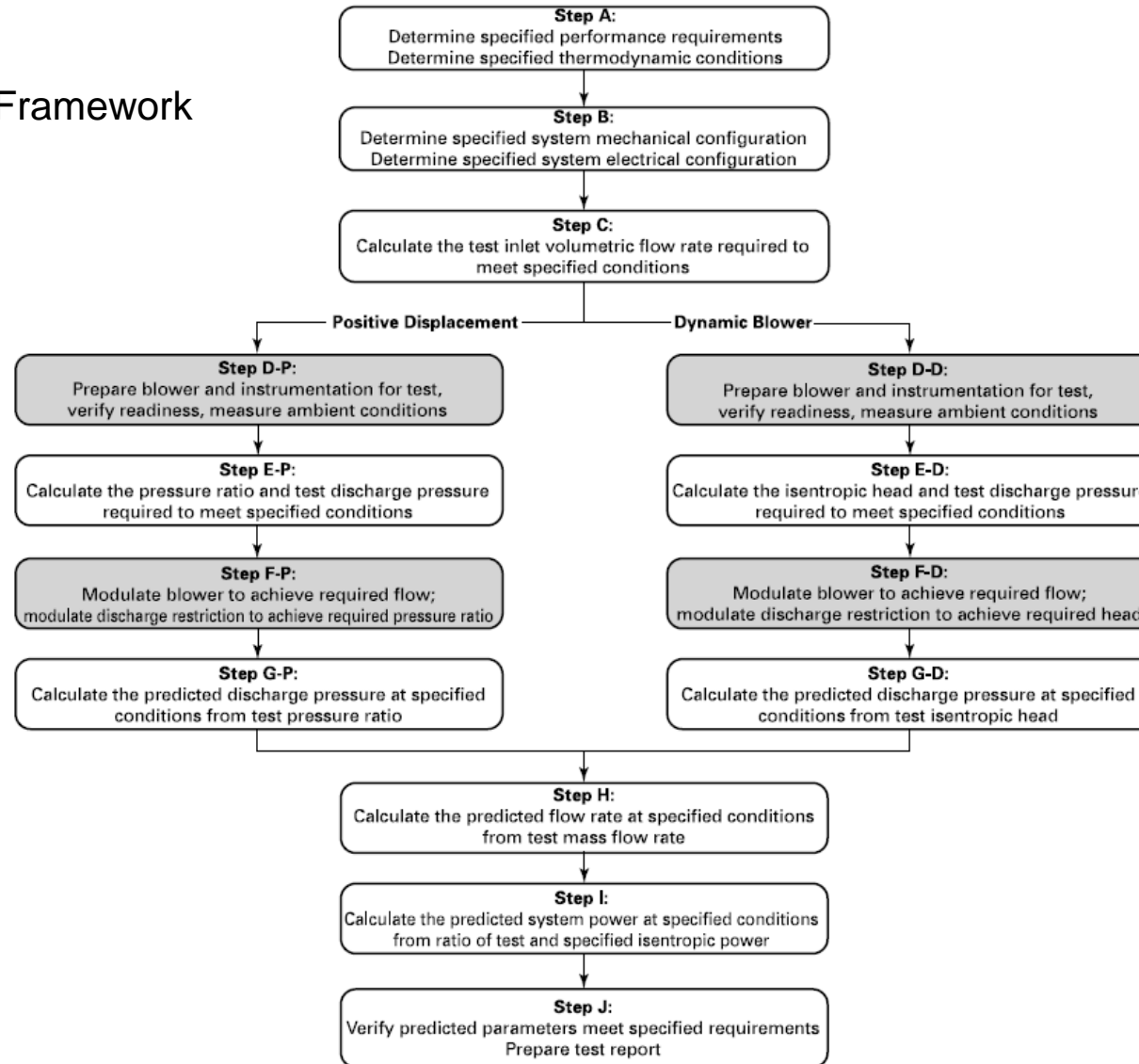
# ASME PTC 13 Testing Philosophy - Methodology

---

- Adjustment of Test Conditions:
  - Match test conditions to specified parameters, emphasizing flow rate and isentropic head/pressure ratio.
- Testing Methodology:
  - Dynamic (Centrifugal) Blowers: Match operating speed with inlet flow rate and isentropic head, using additional flow control methods as needed.
  - Positive Displacement (PD) Blowers: Match operating speed with inlet flow rate and pressure ratio.
- Guidance on Deviations:
  - Detailed directives for acceptable variations in operating conditions and air properties.

# Calculation Methodology Framework

Figure 5-1.1-1  
Calculation Methodology Framework



Included in test

Determine by calculation

# Specified Performance Test Points and Conditions

Table 2 Continuous Duty Operating Conditions					
100 Hp Blowers					
Operating Condition <sup>(1)</sup> (2)	Inlet Temperature (deg. F)	Inlet Relative Humidity (%)	Discharge Airflow Rate (scfm)	Discharge Pressure (psig)	Total Wire Power (kW)
Condition 1	95	80%	1,850	6.5	63.5
Condition 2	95	80%	665	6.5	34.1
Condition 3	85	80%	1,900	6.5	64.0
Condition 4	25	0%	685	6.5	36.2
Condition 5	65	80%	1,960	6.5	N/A
Condition 6	15	0%	725	6.5	N/A
Condition 7	95	80%	1,350	4.75	N/A
Condition 8	95	80%	325	4.75	N/A
Condition 9	15	80%	1,430	1.25	N/A
Condition 10	15	0%	355	1.25	N/A
60 Hp Blowers					
Operating Condition <sup>(1)</sup> (2)	Inlet Temperature (deg. F)	Inlet Relative Humidity (%)	Discharge Airflow Rate (scfm)	Discharge Pressure (psig)	Total Wire Power (kW)
Condition 1	95	80%	1,350	4.75	42.1
Condition 2	95	80%	325	4.75	17.2
Condition 3	85	80%	1,390	4.75	43.2
Condition 4	25	0%	335	4.75	19.0
Condition 5	65	80%	1,430	4.75	N/A
Condition 6	15	0%	355	4.75	N/A
Condition 7	15	80%	1,430	1.25	N/A
Condition 8	15	0%	355	1.25	N/A

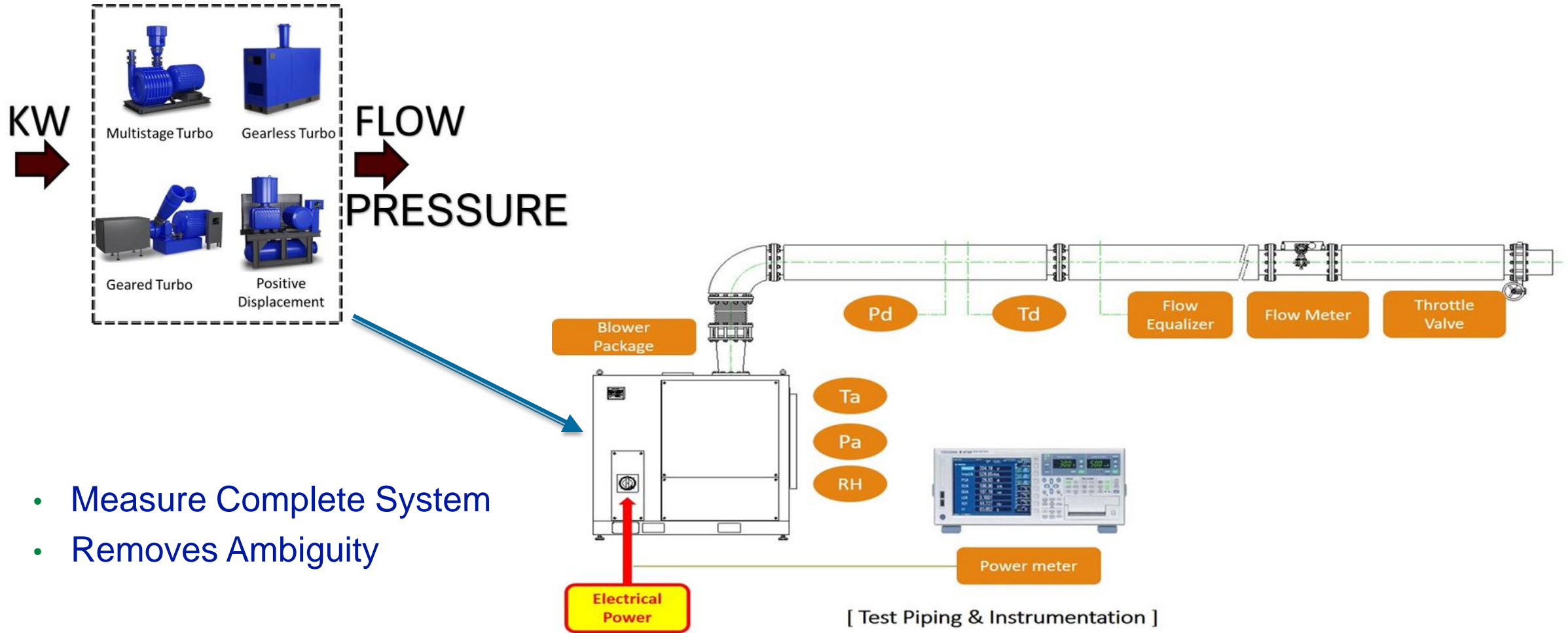
Notes:

(1) Each operating condition shall be considered with the discharge pressure and flow measured downstream of the check valve and shall include inlet losses as noted in Table 1.

(2) The blowers shall be capable of continuously operating between Operating Conditions listed. The Total Wire Power performance shall be guaranteed at Conditions 1, 2, 3, and 4 for each blower with no positive tolerance.

- During Testing
  - Establish performance envelope as specified
  - Measure blower performance at each point at test conditions
  - Calculate predicted blower performance at specified site conditions
  - Evaluate results

# Typical PTC 13 Blower Test Stand Setup

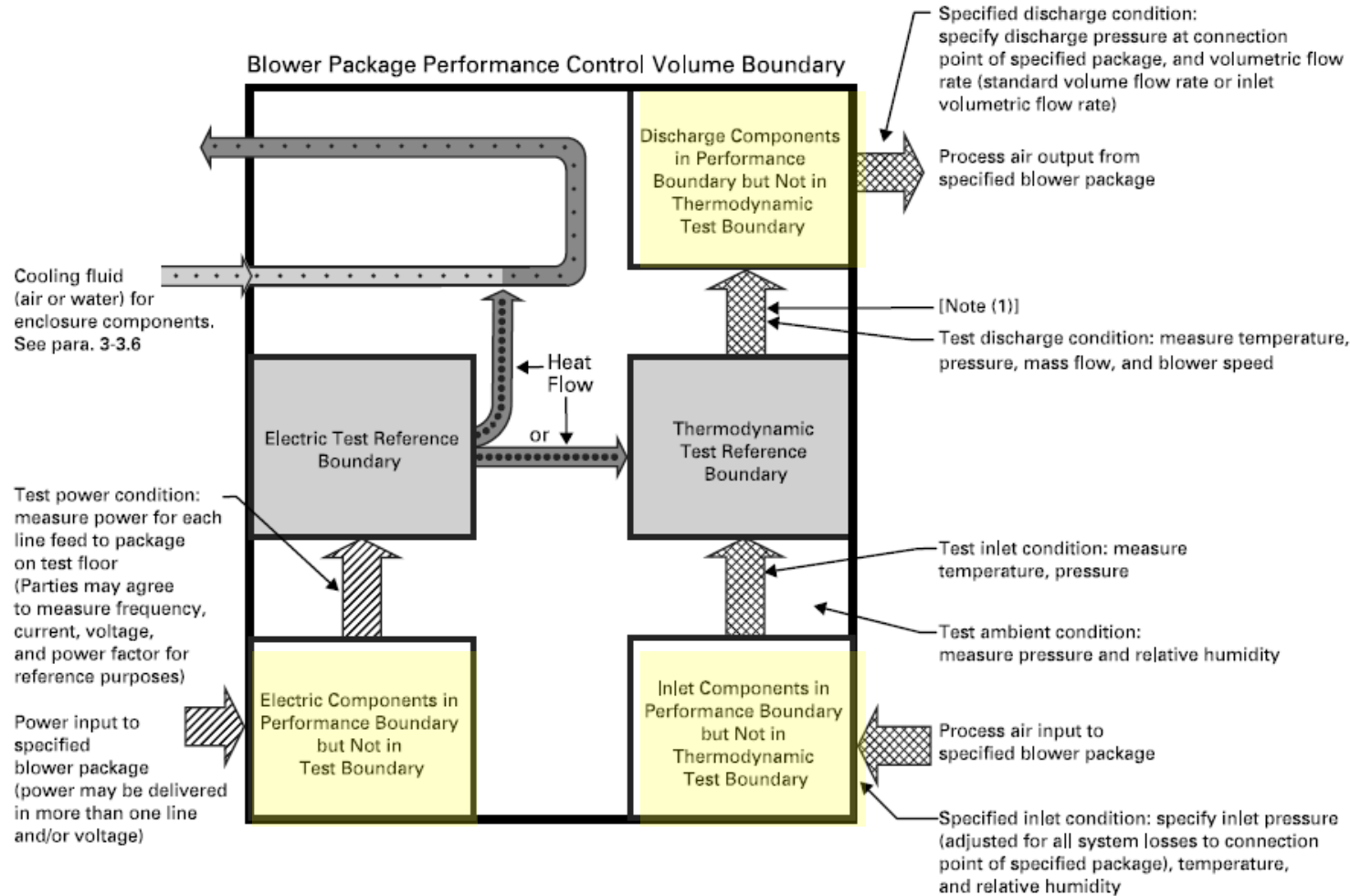


- Measure Complete System
- Removes Ambiguity



# Control Volume for Thermodynamic and Electrical Reference Boundary

Figure 3-5.2-1 Blower Package Performance Boundary and Internal Reference Boundaries



Included in test

Determine by calculation

**NOTE:**

(1) Test discharge operating condition: set blower condition to match inlet volumetric flow at specified condition for setting pressures.

(a) Determine, by calculation, specified inlet pressure to account for inlet components not in test boundary.

(b) Determine, by calculation, specified discharge pressure to account for discharge components not in test boundary.

(c) Set pressure to match specified isentropic head (centrifugal) or to match specified pressure ratio (displacement).

See [Section 3-2](#).

# Specify Thermodynamic Reference Boundary

Table 3-5.2-1 – Process and Fluid Components

		Included in Performance Boundary		
		Included in Test	Determine by Calculation	Not Applicable
a	Inlet filter	X		
b	Inlet silencer	X		
c	Discharge silencer			X
d	Inlet isolation valve			X
e	Throttling valve			X
f	After cooler			X
g	Misc. pipe and fittings	X		
h	Inlet air cooler			X
i	Discharge check valve			X
j	Discharge isolation valve			X
j	Enclosure doors or panel openings	X		
l	Estimated system inlet press drop		X	
k	Additional components not listed included as forming the blower Package			X

# Specify Electrical Power Reference Boundary

Table 3-5.2-2 – Electrical Power Related Components

		Included in Performance Boundary		
		Included in Test	Determine by Calculation	Not Applicable
a	Drive motor	X		
b	Motor cooling fan(s)	X		
c	Magnetic Bearing and Controller			X
d	Bearing cooling fan(s)			X
e	Coolant Pumps			X
f	Lubrication Pumps and Accessories			X
g	Heat Exchanger Fans	X		
h	Package Cooling Fan	X		
i	VFD	X		
j	VFD Line Side Power conditioning Equipment			X
j	VFD Load Side Power conditioning Equipment	X		
l	Eddy Current or Variable Speed Clutch			X
k	Operation Control Panel(s)	X		
l	Power/isolation transformers and power supplies	X		
m	Power conditioner			X
n	Blower and motor cooling	X		
o	VFD cooling	X		
p	Additional components not listed included as forming the blower package			X

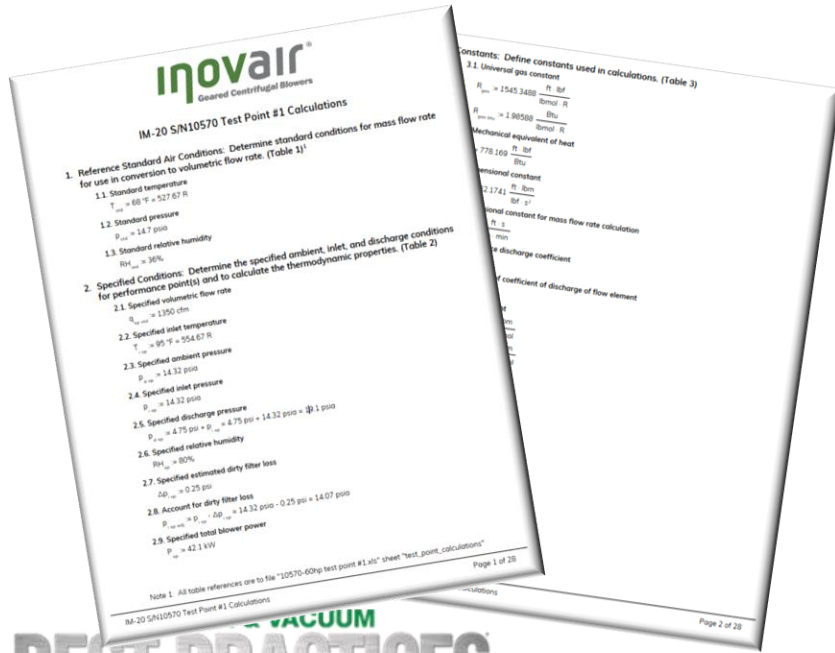
# Establishing Performance Points, Calculations and Reporting

---

- Follow PTC-13 calculation procedures:
  - Section 5 - Computation of Results
  - Section 6 - Report of Results
  - *Section 7 - Test Uncertainty*
  - Mandatory Appendix I - Airflow Conversions
  - Nonmandatory Appendices
    - A - Sample Uncertainty Calculations
    - B - Sample Calculation: Dynamic Blowers
    - C - Sample Calculation: PD Blowers
    - D - Sample Calculation: Machine Reynolds Number Correction
    - E - Sample Calculation: Orifice Flowmeter

# Developing a Test Plan

- Develop test plan with test report template
- Develop Test point calculation Sheets
- Develop sample calculations
- Conduct pretest



**inovair**  
Geared Centrifugal Blowers

Temperature	$T_{std}$	68.0 °F
Pressure	$P_{std}$	14.70 psia
Relative Humidity	$RH_{std}$	36%

Volumetric Flow Rate	$Q_{vol}$	1.850 SCFM
Inlet Temperature	$T_{in}$	95.0 °F
Barometric Pressure	$P_{bar}$	14.32 psia
Inlet Pressure	$P_{in}$	14.32 psia
Discharge Pressure	$P_{out}$	6.50 psig
Relative Humidity	$RH_{in}$	80%
Estimated Dirty Filter Loss	$\Delta P_f$	0.25 psi
Account for Dirty Filter Loss	$P_{in,adj}$	14.07 psia
Total Power	$P_{tot}$	63.5 kW

Universal Gas Constant	$R_{univ}$	1545.3488 ft·lbf/(lbmol·°R)
Mechanical Equivalent of Heat	$J$	778.169 ft·lbf/Btu
Temperature Conversion, °F to °R		459.67
Proportionality Constant	$K_c$	32.1741 lbm·ft/(lbf·sec <sup>2</sup> )
Units Conversion Constant	$K_{uc}$	5.0 ft·sec/(in·min)
Orifice Discharge Coefficient	$C_d$	0.60 (initial value)
Uncertainty of Discharge Coefficient	$U_c$	0.60%
Molecular Weight of Dry Air	$MW_{da}$	28.970 lbm/lbmol
Molecular Weight of Water Vapor	$MW_{wv}$	18.015 lbm/lbmol

Inlet Pipe Inner Diameter	$D_{i1}$	9.35 in
		0.779 ft
Inlet Pipe Inner Area	$A_{i1}$	0.48 in <sup>2</sup>
Assumed Efficiency	$\eta$	
Assumed Ratio of Specific Heats	$\kappa$	
Orifice Diameter	$d_{orifice}$	
Measured Discharge Pipe Inner Diameter	$D_{meas}$	
Measured Discharge Pipe Inner Area	$A_{meas}$	
Pipe & Orifice Measurement Temperature	$T_{meas}$	
Orifice Plate Coefficient of Expansion	$F_{exp}$	
RTD Recovery Factor	$r_f$	
Motor Nameplate Base Speed	$N_b$	
Gearbox Ratio	$R_g$	
Pulley Ratio	$R_p$	
Total Ratio	$R_t$	
Impeller Diameter (Exducer)	$D_{impeller}$	

Ambient Temperature	$T_{amb}$	83.0 °F
Barometric Pressure	$P_{bar}$	14.32 psia
Relative Humidity	$RH_1$	44.30 %

Saturated Vapor Pressure Beta Coefficients	$\beta_{wv,amb}$	-15.7516
	$\beta_{wv,in}$	-14.8733
	$\beta_{wv,out}$	-15.2005

Pressure Ratio	$r_{p,spec}$	1.480
Air Mixture Molecular Weight at Specified Condition	$MW_{mix,spec}$	0.064
Air Mixture Density at Specified Condition	$\rho_{mix,spec}$	0.0673 lbm/ft <sup>3</sup>
Mass Flow of Mixture	$\dot{m}_{mix,spec}$	142.3 lbm/min
Inlet Velocity at Specified Condition	$V_{in,spec}$	4.9312 ft/min
Inlet Total Temperature at Specified Condition	$T_{in,tot,spec}$	551.7 °R
Pressure Ratio (Total Pressure)	$r_{p,tot,spec}$	1.480
Molar Specific Heat of Air	$C_{p,air,mol}$	6.951 Btu/lbmol·°R
Molar Specific Heat of Vapor	$C_{p,wv,mol}$	8.029 Btu/lbmol·°R
Specific Heat of the Air Mixture	$C_{p,mix}$	0.248 Btu/lbm·°R
Ratio of Specific Heats of the Air Mixture	$\kappa_{mix}$	1.396
Isentropic Head at Specified Condition	$W_{is,spec}$	12.482 ft·lbf/lbm
Isentropic Power at Specified Condition	$P_{is,spec}$	48.1 kW

Estimated Pressure Ratio	$r_{p,est}$	1.480
Air Mixture Molecular Weight at Test Condition	$MW_{mix,est}$	0.0584
Estimated Discharge Pressure	$P_{out,est}$	20.80 psia
Estimated Temperature Ratio for the Test Conditions	$r_{t,est}$	1.582
Estimated Discharge Temperature for the Test Conditions	$T_{out,est}$	643.4 °R
Molar Specific Heat of Air	$C_{p,air,mol}$	6.948 Btu/lbmol·°R
Molar Specific Heat of Vapor	$C_{p,wv,mol}$	8.021 Btu/lbmol·°R
Specific Heat of the Air Mixture	$C_{p,mix,est}$	0.242 Btu/lbm·°R
Ratio of Specific Heats of the Air Mixture	$\kappa_{mix,est}$	1.399
Target Discharge Pressure	$P_{out,target}$	21.13 psia
Estimated Discharge Air Density	$\rho_{out,est}$	0.0680 lbm/ft <sup>3</sup>
Estimated Discharge Air Velocity	$V_{out,est}$	1.931 ft/min
Target Discharge Pressure	$P_{out,target}$	21.12 psia
		7.00 psig

Static Discharge Pressure of Test gauge	$P_{out,static}$	7.00 psig
Discharge Air Density	$\rho_{out}$	0.0680 lbm/ft <sup>3</sup>
Inlet Total Temperature	$T_{in,tot}$	544.2 °R
Inlet Total Pressure	$P_{in,tot}$	14.32 psia
Inlet Total Air Density	$\rho_{in,tot}$	0.0694 lbm/ft <sup>3</sup>
Discharge Total Temperature	$T_{out,tot}$	645.32 °R
Discharge Total Pressure	$P_{out,tot}$	185.65 °F
Discharge Total Air Density	$\rho_{out,tot}$	0.0680 lbm/ft <sup>3</sup>
Office Air Density	$\rho_{office}$	0.0821 lbm/ft <sup>3</sup>
Absolute Velocity, Dry Air	$V_{abs,da}$	4.39E-07 ft/sec/ft <sup>3</sup>
Absolute Velocity, Water Vapor	$V_{abs,wv}$	2.37E-07 ft/sec/ft <sup>3</sup>
Absolute Velocity, Mix	$V_{abs,mix}$	1.40E-05 ft/sec/ft <sup>3</sup>

Test Orifice Diameter	$d_{orifice}$	5.023 in
Test Discharge Pipe Inner Diameter	$D_{meas}$	12.396 in
Test Ratio of Orifice and Pipe Diameter	$\beta$	0.404
Test Discharge Pipe Inner Area	$A_{meas}$	0.838 in <sup>2</sup>
Orifice Discharge Coefficient	$C_{d,est}$	0.60141
Expansion Factor	$F_{exp}$	0.985
Mass Flow of Test	$\dot{m}_{test}$	146.4 lbm/min
Volumetric Flow Rate of Test	$Q_{vol,test}$	2.384 cfm
Inlet Air Velocity	$V_{in}$	4.413 ft/min
Discharge Air Velocity	$V_{out}$	1.986 ft/min
Orifice Air Velocity	$V_{orifice}$	1.982 ft/min
Reynolds Number	$Re_d$	2.15E+05
Orifice Discharge Coefficient, Calculated	$C_{d,calc}$	0.60141
Orifice Discharge Coefficient, % Difference		0.001%
2% of Uncertainty		0.012%
% Difference < 2% of Uncertainty	Yes	

Pressure Ratio (Total Pressure)	$r_{p,tot}$	1.300
Isentropic Head at Test Conditions	$W_{is}$	12.366 ft·lbf/lbm
Isentropic Power at Test Conditions	$P_{is}$	41.6 kW

Predicted Discharge Temperature at Specified Conditions	$T_{out,spec}$	653.5 °R
Predicted Discharge Air Density at Specified Conditions	$\rho_{out,spec}$	0.0845 lbm/ft <sup>3</sup>
Specific Volume Ratio at Specified Conditions	$r_{v,spec}$	1.214
Specific Volume Ratio at Test Conditions	$r_{v,test}$	1.285
Limit of 95 to 105%	$r_{v,95-105}$	100.7%
Inlet Volumetric Flow Rate at Specified Conditions	$Q_{vol,spec}$	2.113 cfm
Inlet Volumetric Flow Rate at Test Conditions	$Q_{vol,test}$	2.384 cfm
Limit of 98 to 102%	$Q_{vol,98-102}$	99.4%
Isentropic Head at Specified Condition	$W_{is,spec}$	0.282 ft·lbf/lbm
Isentropic Head at Test Conditions	$W_{is,test}$	0.224 ft·lbf/lbm
Limit of 100 to 105%	$W_{is,100-105}$	100.7%
Impeller Speed	$N_1$	34.365 RPM
Machine Mesh Number at Specified Conditions	$M_{mesh,spec}$	0.726
Machine Mesh Number at Test Conditions	$M_{mesh,test}$	0.736
Refer to Chart for Limits	$M_{mesh,spec} - M_{mesh,test}$	0.010

Predicted Discharge Pressure for the Specified Conditions	$P_{out,est}$	20.87 psia
Predicted Volumetric Flow Rate for Standard Conditions	$Q_{vol,est}$	6.50 psig
Limited from 98 to 102%	$Q_{vol,98-102}$	3.842 cfm
Predicted System Power for Specified Conditions	$P_{tot,est}$	70.0 kW
Predicted as Percent of Specified Power		110.0%
Entire Blower System Efficiency	$\eta_{tot,est}$	57.3%
Blower Efficiency	$\eta_{blower}$	85.9%

**inovair**  
Geared Centrifugal Blowers

**BEST PRACTICES**  
blowervacuumbestpractices.com

# Witness Test Roles and Responsibilities

## Role and Responsibilities of the Test Witness

- Specify testing requirements
- Review and approve test plan
- “Bear witness” the test was conducted in conformance with purchase contract requirements
- Ascertain that the proposed test plan meets contract requirements and the requirements of ASME PTC 13
- Make observations to document that the test facilities and conduct of the test are in accordance with contract and ASME PTC 13
- Review the final report from the testing entity to evaluate the reported results represent the experience of the test
- Exercise an opinion as to whether the test results indicate that the blower equipment has met contractual requirements

## Role and Responsibilities of the Test Entity

- Prepare a detailed test plan that describes facilities and equipment, methods of testing, predictive calculations and calculation aids, proposed allowable tolerances in test results, and final reporting examples.
- Provide all the equipment, personnel, resources, and logistics to conduct the tests described in the test plan.
- Operate all the equipment involved in the test.
- Collect and record all data.
- Provide a comprehensive report of the test and the site performance results predicted through application of ASME PTC 13.

# Inovair Blower Test Stand



- Insulated blower discharge and flow measurement station
- Separate test room with multiple computer screens



- Blower with inlet measurement station
- VFD with control panel



# Things Inovair did well and what can be improved

---

- Inovair prepared a detailed test plan, test report template and sample calculation sheets
- Sample calculation were submitted with the test protocol in advance
- Multiple resubmittals were required with the test protocol. Not as many with the test report
- Blowers were mechanically pretested and ran in a stable manner during test
- Inovair blower components are manufactured onsite. Other components are sourced locally
- Inovair was able to manufacturer blower components and retest in a week
- Pretest must include performance testing and verification of the test points
- Inovair needs update blower performance information based on ASME PTC 13 testing



# Inovair Blowers

## 2 Stacked Units



## Single Unit with VFD



# Test Protocol/Procedure Submittal

---

- Review test protocol submittal at least 30 days in advance
- Schedule meeting with vendor to expedite review if needed
- Submittal should include:
  - Calibration certificates of all instruments to be used dated within previous 12 months
  - Points to be tested – guarantee points and 5-point curve if required
  - Test arrangement including flow meter straight runs
  - Flow meter information
  - Data collection sheet
  - Rotor balance reports
- Several resubmittals were required. Will go more smoothly next time.
- Inovair submitted sample PTC13 calculation which expedited the report submittal. (Recommend requiring sample calculation of one test point for thermodynamic condition correction)

# Witness Testing – Performance Guarantees

Vendor shall guarantee the following power numbers for the operating points listed below.

Performance Table									
	A	B	C	D	E	F	G	H	I (G x H)
	Temp (°F)	RH (%)	Total Inlet Flow per Blower (ICFM) <sup>1</sup>	Total Inlet Flow per Blower (SCFM)	Blower Discharge Pressure (psig)	Blower Inlet Pressure Losses (psi)	Guaranteed Power per Blower <sup>2</sup> (kW)	Weight Factor (%)	Factored Power per Blower (kW)
1	100	95	1,754	1,533	9.4	0.3	64.2	15	9.6
2	68	80	1,582	1,533	9.4	0.3	62.0	50	31.0
3	100	95	1,315	1,150	9.4	0.3	47.5	10	4.8
4	68	80	1,186	1,150	9.4	0.3	46.5	25	11.6
								TOTAL	57.0

Notes:

- Flow shall be measured on the discharge side during testing and corrected to ICFM.
- Guaranteed power consumption indicated shall be the total wire-to-air power consumption of each blower and shall account for any and all losses including, but not limited to, harmonic filters, motor inefficiencies, AFD inefficiencies, cooling fans or cooling pumps, bearing power requirements, and blower controls.
- Performance testing shall test for each specific blower operating point to demonstrate conformance.

2-3. ACCEPTABLE MANUFACTURERS. Blower shall be manufactured by Inovair without exception:

# Witness Testing

## Blower on Test Stand



## VFD on Test Stand



# Witness Testing – How to Conduct

---

- Witness any required onsite instrument calibration or calibration verification
- Verify calibration certificates are available for all instruments & that they match protocol submittal
- Check test stand arrangement to verify it matches protocol submittal
- Review points to be taken and number of readings at each point
- Record test instrument information and verify adequacy including instrument mfr, model, serial, range, division
- Record blower and VFD serial numbers
- Verify location of power measurement & which items are included
- Check route for local direct instrument readings if required

# Witness Testing – How to Conduct, cont'd

---

- Verify thermal stability before starting each reading: No more than 1 deg F change in differential temperature within 5 min
- Start test and timer for readings
- Witness local direct readings and data acquisition readings
- Compare local with DAQ readings
- For centrifugal machines, verify rise to surge requirements are being met
- Take screen shots of DAQ screen
- Request copies of DAQ readings
- If the vendor requires a signature, it should be to verify the readings recorded were the readings witnessed
- Do not sign indicating the unit has passed until there is time to review test report independently in office

# Witnessing & Discussion of Readings in Test Lab

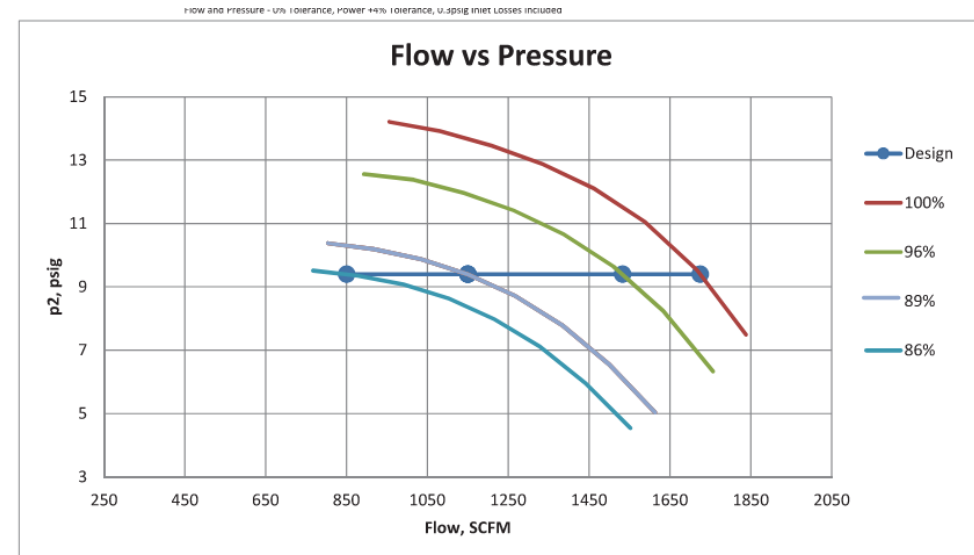


# Witness Testing

## What to Look for

- While moving up curve, pressure should increase and flow decrease
- Lack of stability between readings at same point
- Objectionable vibration
- Objectionable temperature
- Objectionable noise
- Overhead door opening
- Cooling fan starting or stopping

## Example Blower Curve





# Test Report & Presentation of Results

---

- Check sample calculation vs specified site ambient conditions, recorded test lab conditions and PTC13 for methodology
- Compare DAQ readings submitted with screen shots or copies provided at time of test
- Spot check of witnessed local direct instrument values vs values in test report
- Compare local direct readings with DAQ readings (this process should be started during the test)
- Verify required curves have been submitted and that serial numbers are indicated
- Recheck rise to surge requirements are met

# Evaluation & Acceptance of Results

---

- Check flow, pressure, and power vs requirements
- Verify whether power guarantees are met by including weighting factors, present worth factor, etc.

# Things Inovair Did Well

---

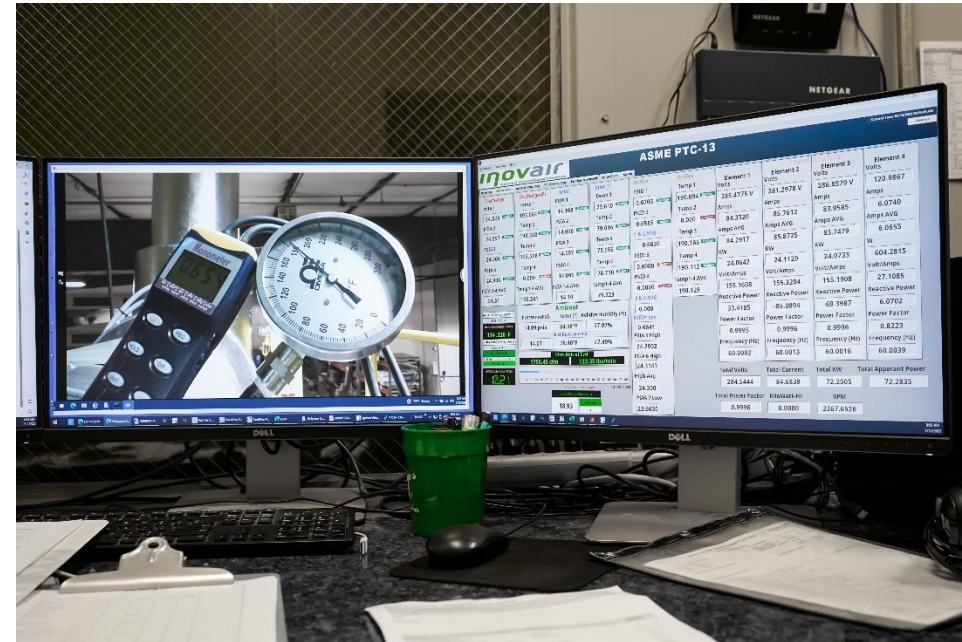
- Blowers ran in a stable manner
- A camera was added to view the instruments on the catwalk from the floor (saved time and enhanced safety)
- Inovair has the capability to modify impellers onsite in case a machine does not perform as required
- Sample calculation was submitted with test protocol in advance of test report
- Initially there were not enough blanks on the data sheet to allow recording of 3 readings at each point but this was quickly corrected

# Catwalk Flow Instruments & View in Test Lab

Catwalk with orifice plate instruments



Camera image of catwalk instruments & DAQ screen



# Data Collection Sheet



Test Point Data Collection & Notes						
Blower Package	Test Point			Date	Time	
Parameter	Symbol	Reading #1	Reading #2	Reading #3	Average Reading	Notes
<b>Ambient</b>						
Barometric Pressure	$P_a$					
Ambient Temperature	$T_a$					
Relative Humidity	$RH$					
<b>Inlet</b>						
Inlet Pressure	$P_i$					
Inlet Temperature	$T_i$					
<b>Discharge</b>						
Discharge Pressure	$P_d$					
Discharge Temperature	$T_d$					
<b>Orifice &amp; Flow</b>						
Orifice Upstream Pressure	$P_o$					
Orifice Downstream Temperature	$T_o$					
Orifice Differential Pressure	$\Delta P_o$					
Volumetric Flow Rate	$Q_v$					
<b>Power</b>						
Motor Speed	$N_{sp}$					
System Power	$P_{sys}$					
Auxiliary Power	$P_{aux}$					
Total Power	$P_{tot}$					
Additional Notes						

# Areas for Improvement

---

- Multiple resubmittals were required for both the test protocol and test report
- Some of the engineering units did not appear in the test report
- Local, direct reading instruments had to be calibrated onsite on the first day of testing. Temperature instruments were time-consuming requiring 8 hrs for everything
- No local relative humidity gauge or pressure upstream of orifice
- Opening/closing of overhead door or running other blowers in test area can affect thermal equilibrium. Inovair was aware of it.
- Most of these issues will be corrected prior to the next test

# Test Instrument Cart

---



# Summary

---

- Before arriving at the manufacturer's shop: Preplanning, reviewing test protocol, pretesting, agreeing on test arrangement, reviewing calibration certificates as well as advance sample calculations (not required but recommended).
- First day at the shop: Performing any needed calibrations, verifying installed instruments and test arrangement.
- All of the above saves time in the long run and results in a better test
- Learning curve for manufacturer and witness: Expect additional time required when a vendor first implements PTC 13 including test protocol submittal, onsite testing, and test report submittal.
- Inovair quickly adjusted to the requirements of PTC13 and a smoother test is anticipated in the future



# The Speakers



**David Sperber P.E.,**  
VP Sales and Marketing



**Glen Roderique**  
Sales Engineer for Geared Centrifugal  
Blowers

Sponsored by

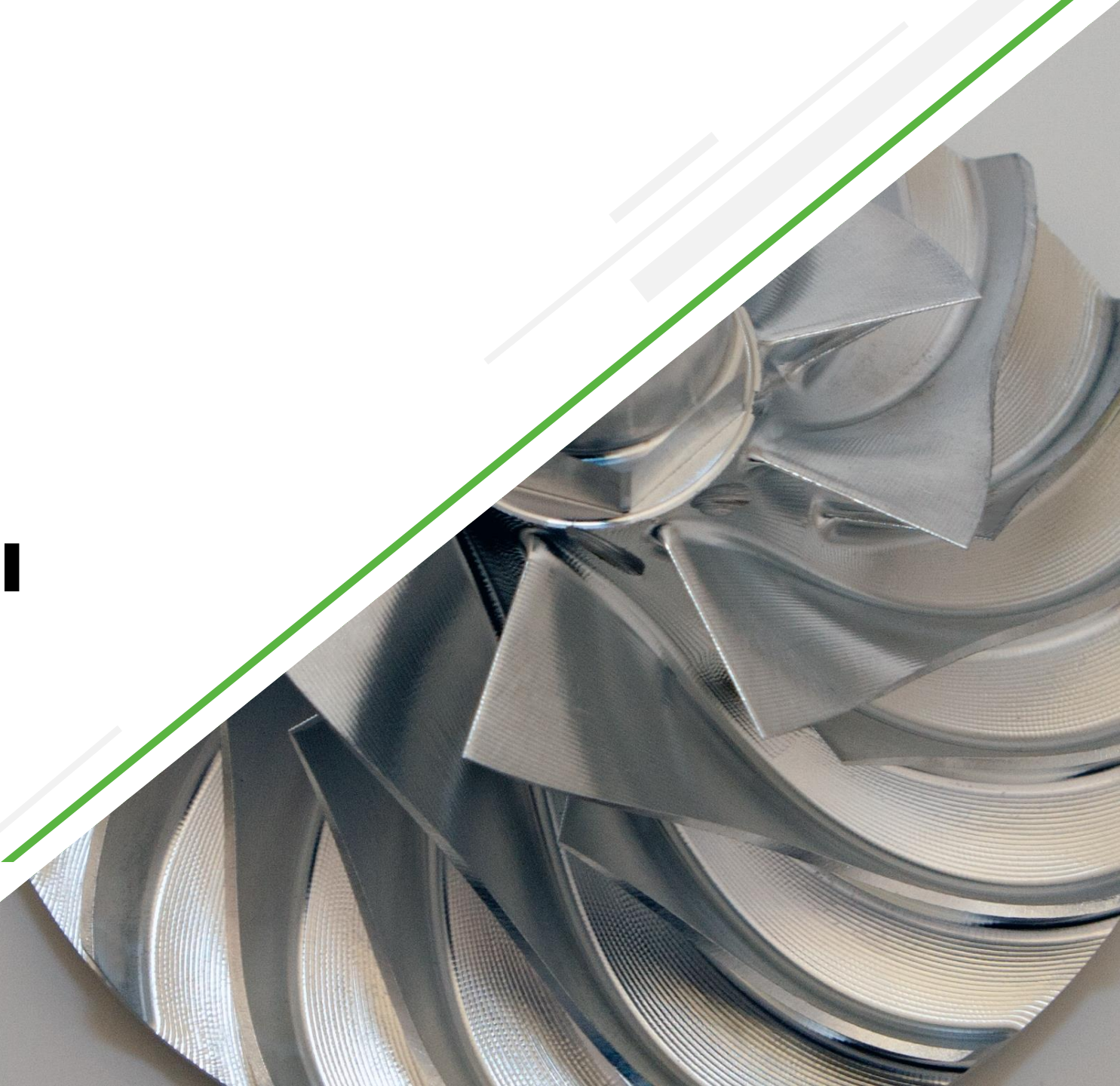


**inovair**<sup>®</sup>  
Geared Centrifugal Blowers

**PTC13 Testing  
as a  
Quality Control Tool**



**PROUDLY MADE  
IN THE USA**



# Blowers Built on a History of Performance

Accessible Technologies Inc. (ATI): the market leader in single stage centrifugal compression for 30 Years

**1994**

**ProCharger  
Superchargers**



**30 Years of High  
Performance - Up to  
75 psi and 6,000 scfm**

**1998**

**Airport  
De-icing**



**Operating in  
Extreme  
Environments**

**2010**

**Continuous Duty  
Industrial**



**Performance in  
24x7 Dirty, Dusty,  
Environment**

**2011**

**Military Ground  
Support**



**99.9% Up-Time  
for Critical  
Operations**

**2012**

**Wastewater**



**Highest Efficiency  
with Leading  
Reliability**

**Over 55,000 Geared Centrifugal Units Sold**

**inovair**<sup>®</sup>  
Geared Centrifugal Blowers




# Vertically Integrated Manufacturer

---

**Inovair is  
“Buy America”  
Compliant**

- Vertically integrated manufacturer with secure U.S. supply chain
- Factory support from **Kansas** four building campus headquarters and **California** regional office with 100 Employees
- Buy America, Build America (BABA) - Manufactured in the U.S. with greater than 55 percent of the total cost of all components from the United States



*Inovair is the only high efficiency  
blower designed, manufactured, and  
supported in the U.S.A.* 

**inovair**<sup>®</sup>  
Geared Centrifugal Blowers

# Inovair PTC 13 Test Cell Implementation

## Challenges of Implementation

- Constructing a near lab-level test cell within our existing facility
- Balancing the size of the system for current and future needs
- Sourcing equipment and instrumentation to meet the standard

## Rewards/Payback of Implementation

- Improving and supplementing our in-house testing capabilities
- Increasing our awareness, focus on entire blower package/system performance
- Providing the capability for us to meet customer project specifications with less uncertainty



Inovair's fully functional PTC 13 test facility

# Blower Testing to Validate Efficiency

## ASME PTC-13

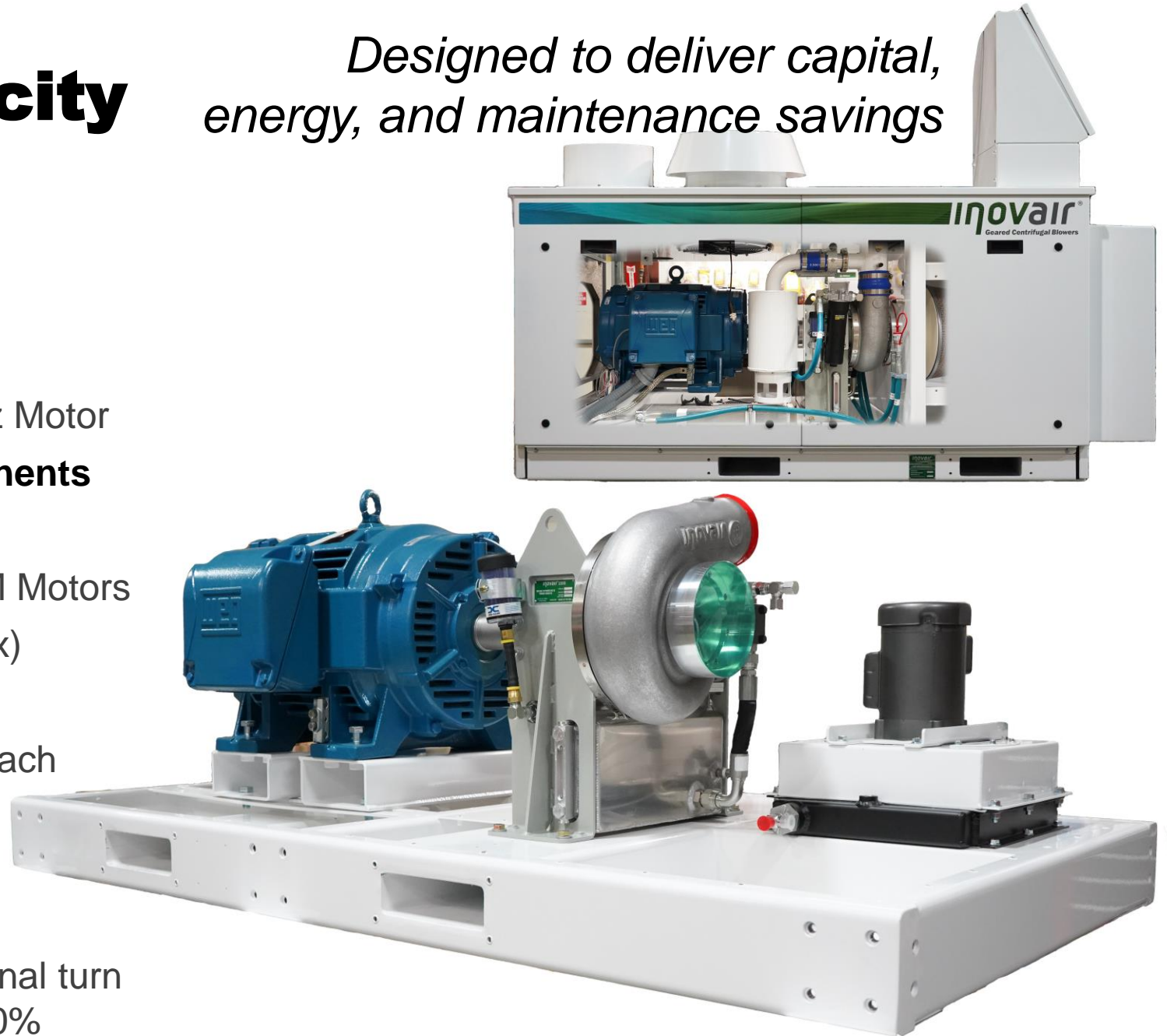
- Method to validate “wire to air” performance for all power components to substantiate total electrical operating power of the blower package
- Testing used to confirm component selection to optimize efficiency
  - Gearbox Selection based on Speed
  - Impeller Selection from 50+ options
  - VFD manufacturer variations
- Blower Component Quality Control
- Inovair staff member has joined the PTC13 committee.



# Design Philosophy Focused on Simplicity

*Designed to deliver capital,  
energy, and maintenance savings*

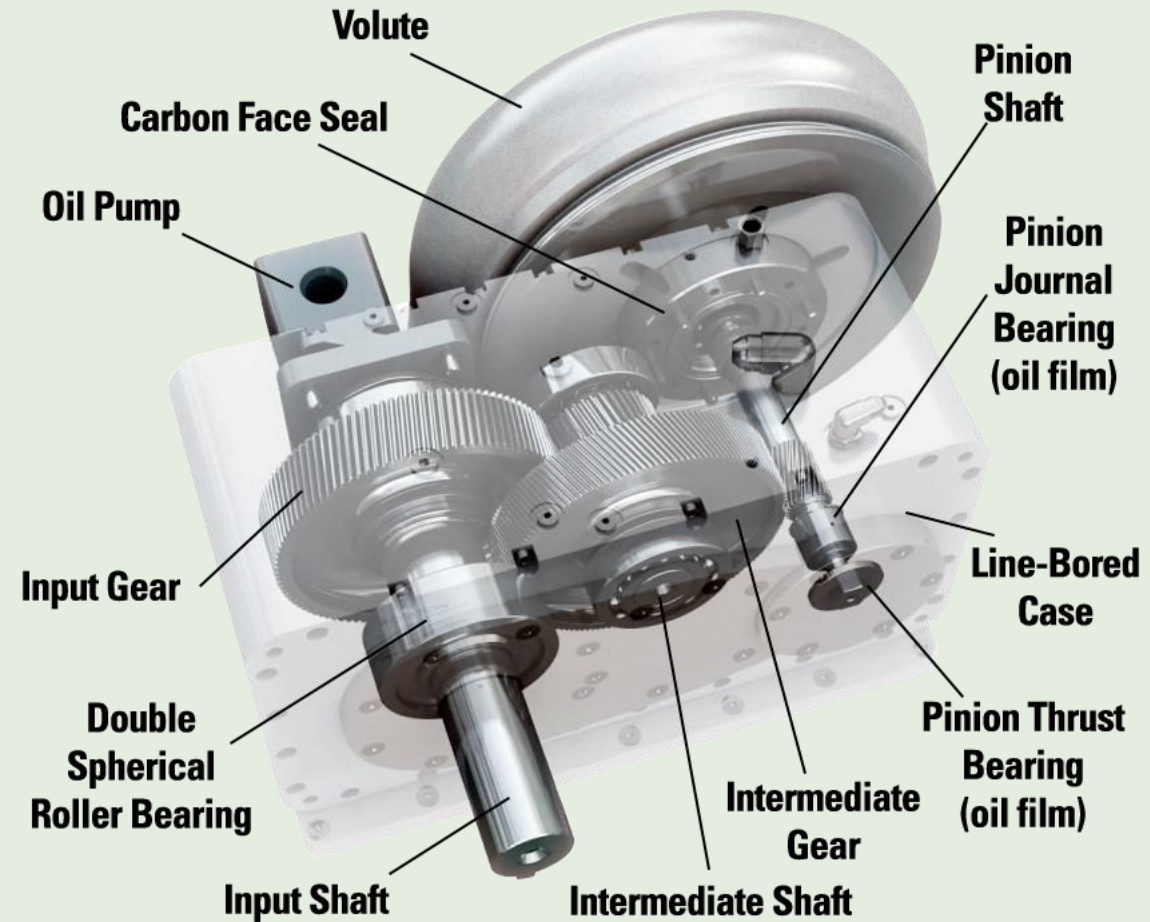
- **Integrally geared**
  - Robust, reliable and cost effective
  - 16.65:1 step-up ratio
- **Speed based control**
  - VFD:1800/3600 RPM TEFC 60 Hz Motor
- **Designed using off the shelf components**
  - 60 Hertz VFD
  - Standard NEMA Frame 1750 RPM Motors
  - PLC (Allen Bradley Compact-Logix)
- **Efficiency**
  - 50+ impeller stages matched for each application
- **Additional Features**
  - Quiet Operation
  - Internal bypass to allow for additional turn down beyond 50%, turndown to 30%



# Integrally-Geared Centrifugal Compressor

- 2-stage gearbox integrated into compressor unit to obtain the designed impeller speed
- Typical operating speeds: 35,000-45,000 rpm
- Designed for up to 22 psi and 60,000 rpm
- Gearboxes rated for up to 600 HP
- High precision gears manufactured to AGMA II standards
- Assemblies balanced to G2.5/G6.3

**inovair**<sup>®</sup>  
Geared Centrifugal Blowers

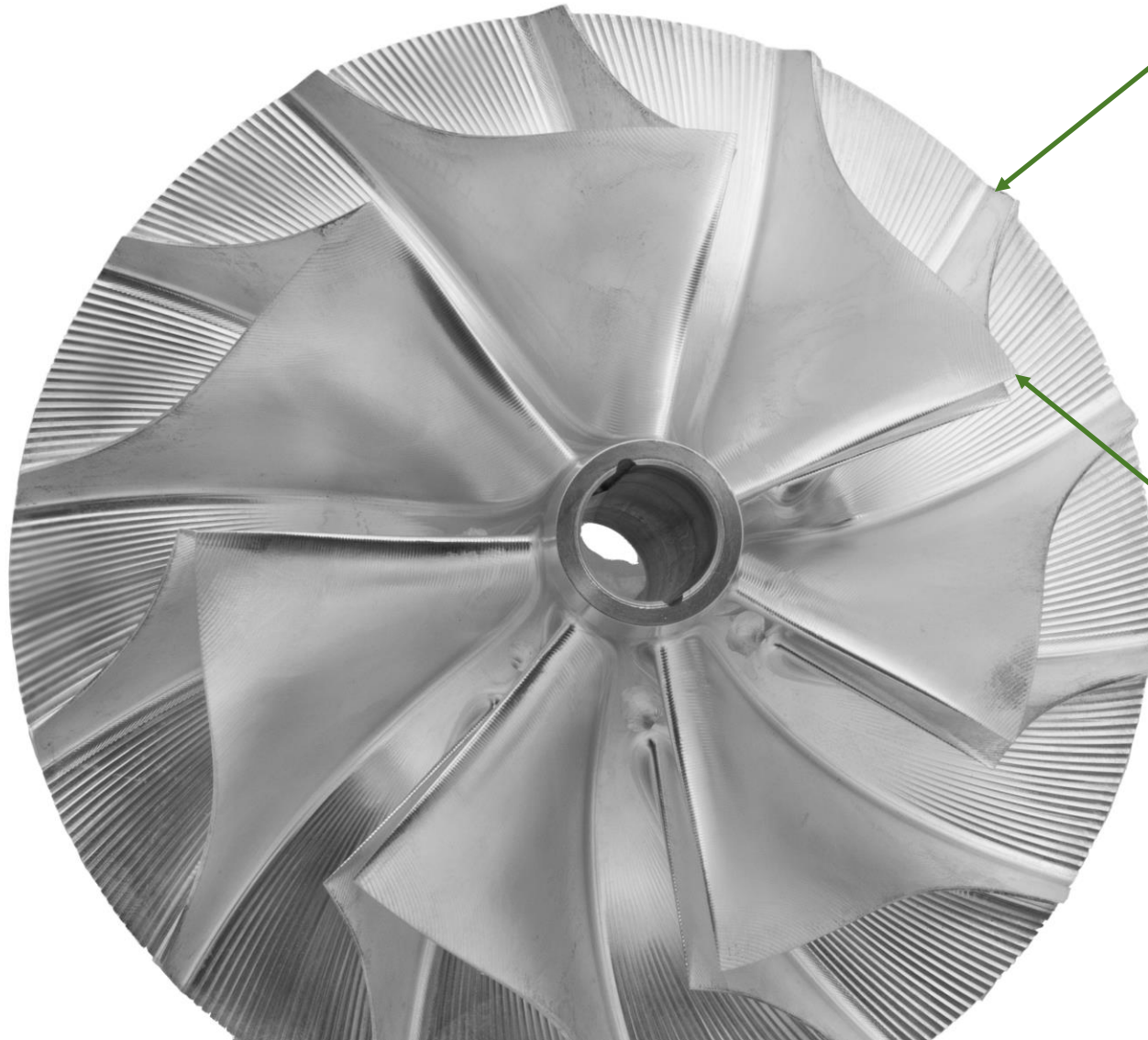


*Designed & manufactured  
in Lenexa, KS*



# High Efficiency Impellers

Proper relationships between exducer diameter and blade height and inducer diameter must be maintained, along with blade geometry, to ensure highest efficiency.



Exducer (outside) diameter and rotational speed (RPM) determine tip speed and thus pressure capability. Pressure increases by a square of the speed.

Inducer diameter and rotational speed effects flow. Flow increases 1:1 with speed.

*Inovair's impellers are machined at the Kansas City campus using one of three 5-axis CNC mills.*



# Versatile Control Systems - constant or varying head applications

- In house UL 508A panel shop
- Standard and custom master control panels
- AB CompactLogix PLC's (custom options available)
- Multiple sensor inputs
- Easily integrates into SCADA or OEM controls
- Accessibility to system programming logic

*Control Panels are engineered and built in-house*

# Process Considerations

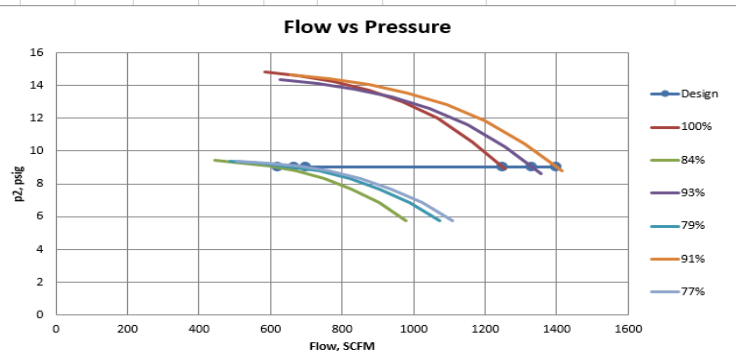
- Airflow Turndown
  - Aeration Basins
    - Airflow Based on Demand
    - D.O. Control
  - Lagoons, MBR's, IFAS Systems



**inovair**  
Efficient • Quiet • Compact

Project: Morris, IL  
Aero Stage: IF2-23-090  
Altitude (ft): 515

Point (#)	Speed (%)	Flow (scfm)	Pressure (psig)	Temp (F)	RH (%)	Motor HP (hp)	System Power (KW)	Design Condition	Isen. Eff. (%)
1	100%	1250	9.00	100	90%	70.2	56.8	Max. Flow @ Max. Temp.	72%
2	84%	620	9.00	100	90%	32.1	26.3	Min. Flow @ Max. Temp.	78%
3	93%	1330	9.00	50	36%	63.7	51.5	Max. Flow @ Avg. Temp.	72%
4	79%	665	9.00	50	36%	29.9	24.5	Min. Flow @ Avg. Temp.	77%
5	91%	1400	9.00	20	20%	63.3	51.2	Max. Flow @ Min. Temp.	72%
6	77%	700	9.00	20	20%	29.4	24.1	Min. Flow @ Min. Temp.	78%



**VS**

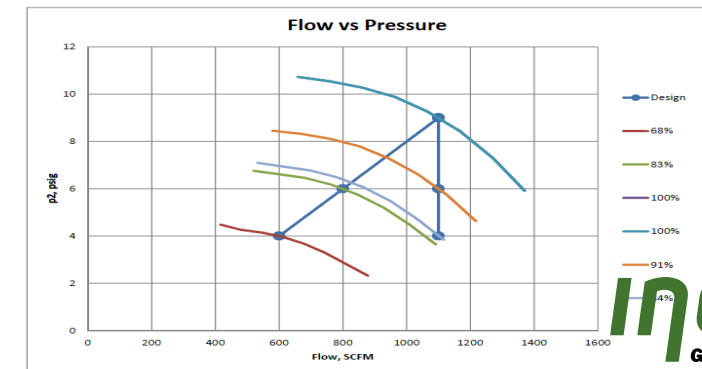
- Varying Water Depths
  - Equalization Tanks
  - Sludge Holding Tanks
  - Aerobic Digesters
  - SBR's



**inovair**  
Geared Centrifugal Blowers

Project: Varying Head  
Aero Stage: IF2-13-100  
Altitude (ft): 0

Point (#)	Speed (%)	Flow (SCFM)	Pressure (PSIG)	Temp (F)	RH (%)	Motor HP (HP)	System Power (KW)	Design Condition
1	68%	600	4.00	100	36%	14.3	12.0	Min. Flow EQ/Digester
2	83%	800	6.00	100	36%	27.4	22.4	Mid Flow EQ/Digester
3	100%	1100	9.00	100	36%	53.4	43.3	Max. Flow EQ/Digester
4	100%	1100	9.00	100	36%	53.4	43.3	Max. Flow SBR
5	91%	1100	6.00	100	36%	40.0	32.6	Mid Flow SBR
6	84%	1100	4.00	100	36%	31.5	25.8	Min. Flow SBR



**inovair**  
Geared Centrifugal Blowers

# Inovair Product Line

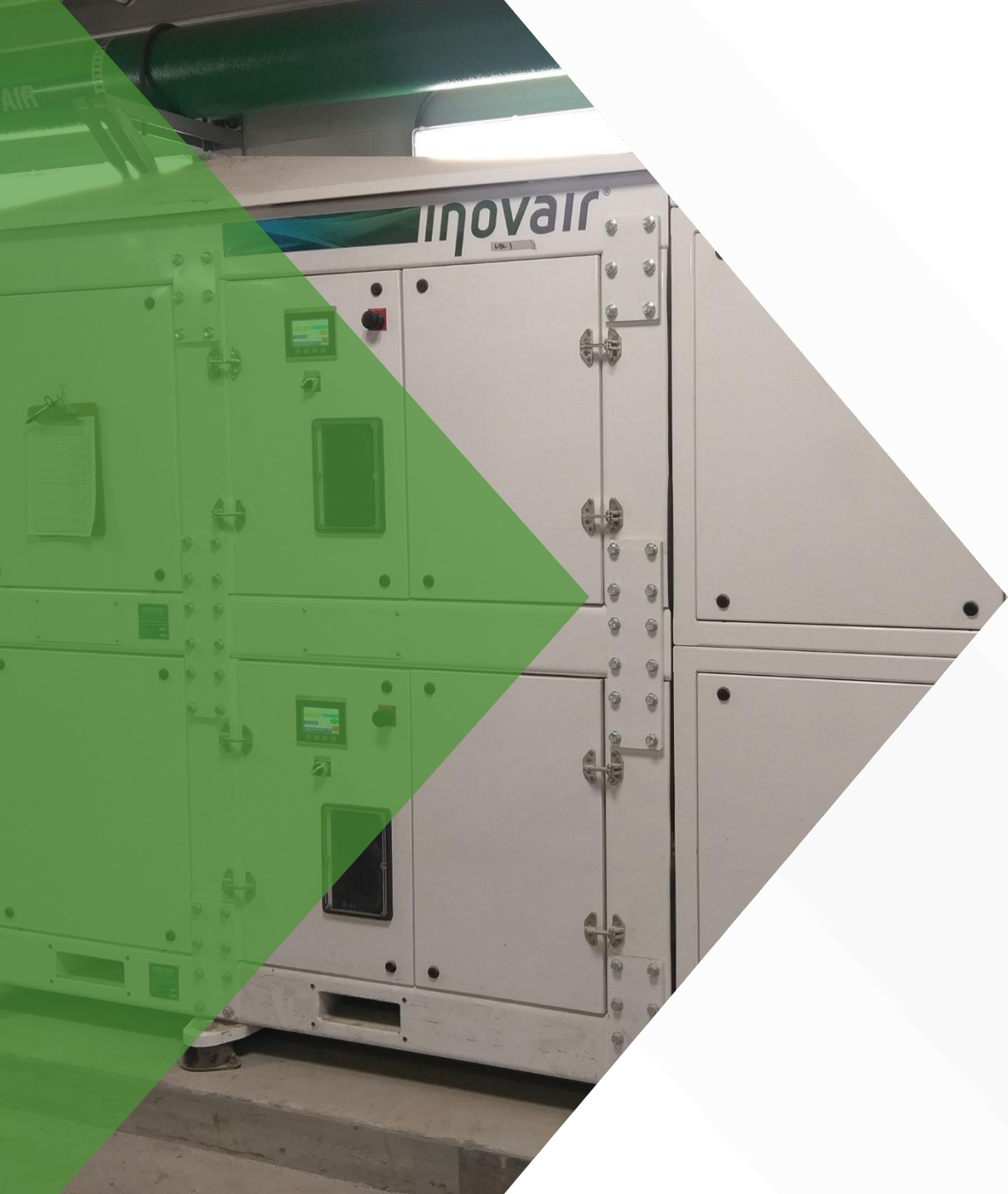
## Design Ranges

- Airflow up to 12,000 SCFM
- Power from 15 hp to 600 hp
- Pressure up to 22 PSI

*Inovair's geared centrifugal blower are designed to deliver capital, energy, and maintenance savings*

## IO/IM SERIES MACHINE LINEUP





***Inovair***<sup>®</sup>  
**Geared Centrifugal Blowers**

**THANK YOU!**

**VISIT US AT  
[INOVAIR.COM](http://INOVAIR.COM)**

**ACCESSIBLE TECHNOLOGIES INC.**



# Speaker Contact Information

---

- Hiran de Mel, P.E., Jacobs email: [Hiran.DeMel@jacobs.com](mailto:Hiran.DeMel@jacobs.com)
- Julie Gass, P.E., Black & Veatch email: [GassJV@bv.com](mailto:GassJV@bv.com)
- David Sperber, P.E., Inovair email: [David.Sperber@inovair.com](mailto:David.Sperber@inovair.com)
- Glen Roderique, Inovair email: [Glen.Roderique@inovair.com](mailto:Glen.Roderique@inovair.com)

---

# ASME PTC-13 in Action: Practical Approach to Blower System Performance Testing

## Q&A

Please submit any questions through the Question Window on your GoToWebinar interface, directing them to Blower & Vacuum Best Practices Magazine. Our panelists will do their best to address your questions and will follow up with you on anything that goes unanswered during this session.

**Thank you for attending!**

Sponsored by



# Thank you for attending!

---

The recording and slides of this webinar will be made available to attendees via email later today.

PDH Certificates will be e-mailed to Attendees by within 2 days



---

## February 2024 Webinar

# Centrifugal vs Rotary Screw Air Compressor Performance: Full Load and Part Load Efficiency



**Mike Lenti**

Compressed Air Consultants  
*Keynote Speaker*

Sponsored by



**Thursday, February 8, 2024– 2:00 PM EST**

Register for free at

[www.airbestpractices.com/webinars](http://www.airbestpractices.com/webinars)